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Lecture abstract

Topics covered in this presentation

- ▶ Steady-state error sources
- ▶ Test waveform inputs for evaluation
- ▶ Sensitivity

Chapter outline

■ 7 Steady-State Errors

- 7.1 Introduction
- 7.2 Steady-state error for unity feedback systems
- 7.3 Static error constants and system type
- 7.4 Steady-state error specifications
- 7.5 Steady-state error for disturbances
- 7.6 Steady-state error for non-unity feedback systems
- 7.7 Sensitivity
- 7.8 Steady-state error for systems in state space

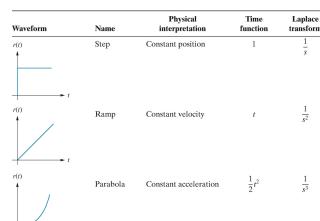
■ 7 Steady-State Errors

■ 7.1 Introduction

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7 Steady-State Errors 7.1 Intro

Some definitions & test waveforms, [1, p. 340]



Steady-state error

- ▶ The difference between the input and the output for a prescribed test input as $t \rightarrow \infty$.

Table: Test waveforms for evaluating steady-state errors of position control systems

Steady-state error in terms of CL TF, $T(s)$, [1, p. 344]

Error between the input, $R(s)$, and the output $C(s)$

$$E(s) = R(s) - C(s)$$

$$C(s) = R(s)T(s)$$

$$E(s) = R(s)[1 - T(s)]$$

Applying the final value theorem

$$e(\infty) = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} sE(s)$$

$$e(\infty) = \lim_{s \rightarrow 0} sR(s)[1 - T(s)]$$

Steady-state error in terms of $G(s)$, [1, p. 345]

Error between the input, $R(s)$, and the output $C(s)$

$$E(s) = R(s) - C(s)$$

$$C(s) = E(s)G(s)$$

$$E(s) = \frac{R(s)}{1 + G(s)}$$

Applying the final value theorem

$$e(\infty) = \lim_{s \rightarrow 0} \frac{sR(s)}{1 + G(s)}$$

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Position constant

$$e_{step}(\infty) = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)}$$

$$K_p = \lim_{s \rightarrow 0} G(s)$$

For a ramp input, $t u(t)$

$$e_{ramp}(\infty) = \frac{1}{\lim_{s \rightarrow 0} sG(s)}$$

$$K_v = \lim_{s \rightarrow 0} sG(s)$$

For a parabolic input, $\frac{1}{2}t^2 u(t)$

$$e_{step}(\infty) = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)}$$

$$K_a = \lim_{s \rightarrow 0} s^2 G(s)$$

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Steady-state error for disturbances, [1, p. 356]

Output

$$C(s) = E(s)G_1(s)G_2(s) + D(s)G_2(s)$$

Error between the input, $R(s)$, and the output $C(s)$

$$E(s) = R(s) - C(s)$$

After substitution

$$\begin{aligned} E(s) &= \frac{1}{1 + G_1(s)G_2(s)}R(s) \\ &\quad - \frac{G_2(s)}{1 + G_1(s)G_2(s)}D(s) \end{aligned}$$

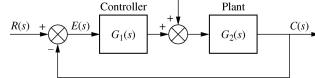


Figure: Feedback control system showing disturbance

Steady-state error for disturbances, [1, p. 356]

Final value theorem

$$e(\infty) = \lim_{s \rightarrow 0} sE(s) = e_R(\infty) + e_D(\infty)$$

Steady-state error due to reference

$$e_R(\infty) = \lim_{s \rightarrow 0} \frac{s}{1 + G_1(s)G_2(s)} R(s)$$

Steady-state error due to disturbance

$$e_D(\infty) = -\lim_{s \rightarrow 0} \frac{s G_2(s)}{1 + G_1(s)G_2(s)} D(s)$$

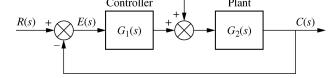


Figure: Feedback control system showing disturbance

Steady-state error due to step disturbance, [1, p. 356]

Steady-state error due to **step** disturbance

$$e_D(\infty) = \frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)}$$

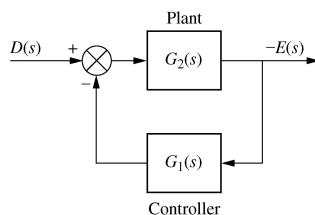


Figure: System rearranged to show disturbance as input and error as output, with $R(s) = 0$

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Sensitivity, [1, p. 362]

Sensitivity: The ratio of the fractional change in the function to the fractional change in the parameter as the fractional change of the parameter approaches zero

$$S_{F:P} = \frac{P \delta F}{F \delta P}$$

Analysis via final value theorem, [1, p. 364]

CL system represented in SS

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx\end{aligned}$$

Error

$$E(s) = R(s) - Y(s)$$

Output

$$Y(s) = R(s)T(s)$$

Substitution

$$E(s) = R(s)[1 - C(sI - A)^{-1}B]$$

Final value theorem

$$\lim_{s \rightarrow 0} sE(s) = \lim_{s \rightarrow 0} sR(s)[1 - C(sI - A)^{-1}B]$$

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Analysis via step input substitution, [1, p. 366]

Steady-state unit step input

$$u = 1$$

Substitution

$$\begin{aligned}0 &= AV + B \\ y_{ss} &= CV\end{aligned}$$

Steady-state solution

$$x_{ss} = V$$

Solving

$$V = -A^{-1}B$$

$$\dot{x}_{ss} = 0$$

Steady-state error

$$\begin{aligned}e(\infty) &= 1 - y_{ss} \\ &= 1 - CV \\ &= 1 + CA^{-1}B\end{aligned}$$

CL system represented in SS

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx\end{aligned}$$

Analysis via ramp input substitution, [1, p. 366]

Steady-state unit step input

$$u = t$$

Substitution

$$\begin{aligned}V &= A(Vt + W) + B \\ y_{ss} &= C(Vt + W)\end{aligned}$$

Steady-state solution

$$x_{ss} = Vt + W$$

Solving

$$\dot{x}_{ss} = V$$

CL system represented in SS

$$V = -A^{-1}B$$

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx\end{aligned}$$

Steady-state error

$$e(\infty) = \lim_{t \rightarrow \infty} t - y_{ss}$$

Bibliography

 Norman S. Nise. *Control Systems Engineering*, 2011.